Hydraulic Dampers

1. HYDRAULIC DAMPERS

1.1. DESCRIPTION

1.1.1 Mode of operation

The pressure fluctuations occurring in hydraulic systems can be cyclical or one-off problems due to:

- flow rate fluctuations from displacement pumps
- actuation of shut-off and control valves with short opening and closing times
- switching pumps on and off
- sudden linking of spaces with different pressure levels.

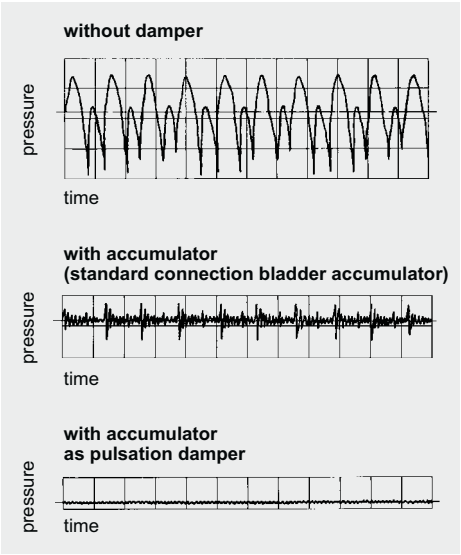
HYDAC hydraulic dampers are particularly suitable for damping such pressure fluctuations.

Selecting the most suitable hydraulic damper for each system ensures that

- vibrations caused by pipes, valves, couplings etc are minimised and subsequent pipe and valve damage is prevented
- measuring instruments are protected and their performance is no longer impaired
- the noise level in hydraulic systems is reduced
- the performance of machine tools is improved
- interconnection of several pumps in one line is possible
- a pump rpm and feed pressure increase is possible
- the maintenance and servicing costs can be reduced
- the service life of the system is increased.

1.2. APPLICATION

1.2.1 Pulsation damping TYPE SB...P / SBO...P



General

The HYDAC pulsation damper

- prevents pipe breaks caused by material fatigue, pipe oscillations and irregular flow rates,
- protects valves, control devices and other instruments,
- improves noise level damping.

Applications

The pulsation damper is particularly suitable for: hydraulic systems, displacement pumps of all types, sensitive measurement and control instruments and manifolds in process circuits in the chemical industry.

Mode of operation

The pulsation damper has two fluid connections and can therefore be fitted directly inline.

The flow is directed straight at the bladder or diaphragm by diverting it in the fluid valve. This causes direct contact of the flow with the bladder or diaphragm which, in an almost inertialess operation, balances the flow rate fluctuations via the gas volume.

It particularly compensates for higher frequency pressure oscillations. The pre-charge pressure is adjusted to individual operating conditions

Construction

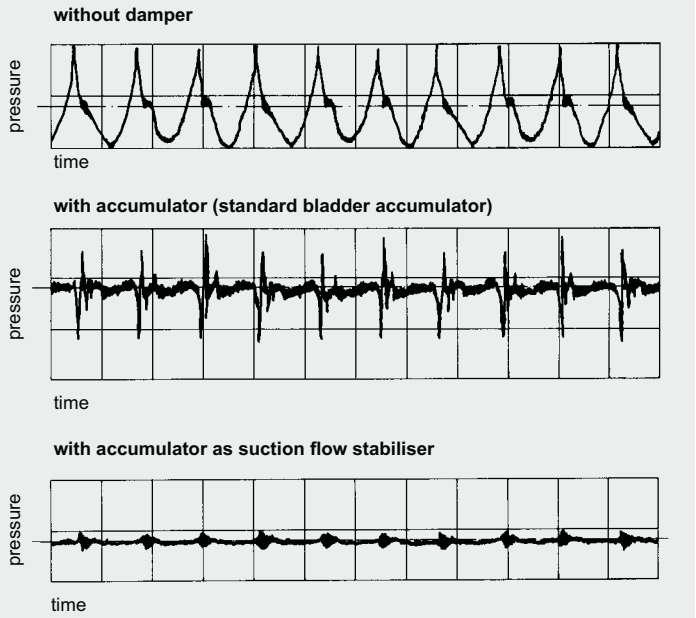
The HYDAC pulsation damper consists of:

- the welded or forged pressure vessel in carbon steel; for chemically aggressive fluids with internal coating or in stainless steel,
- the special fluid valve with inline connection, which guides the flow into the vessel (threaded or flange connection),
- the bladder or diaphragm in various compounds as shown under 1.4.1.

Installation

As close as possible to the pulsation source. Mounting position preferably vertical (gas valve pointing upwards).

1.2.2 Suction flow stabiliser Type SB...S



General
The HYDAC suction flow stabiliser

- improves the NPSH value of the system;
- prevents cavitation of the pump;
- prevents pipe oscillations.

Applications
Main application areas are piston and diaphragm pumps in public utility plants, reactor construction and the chemical industry.

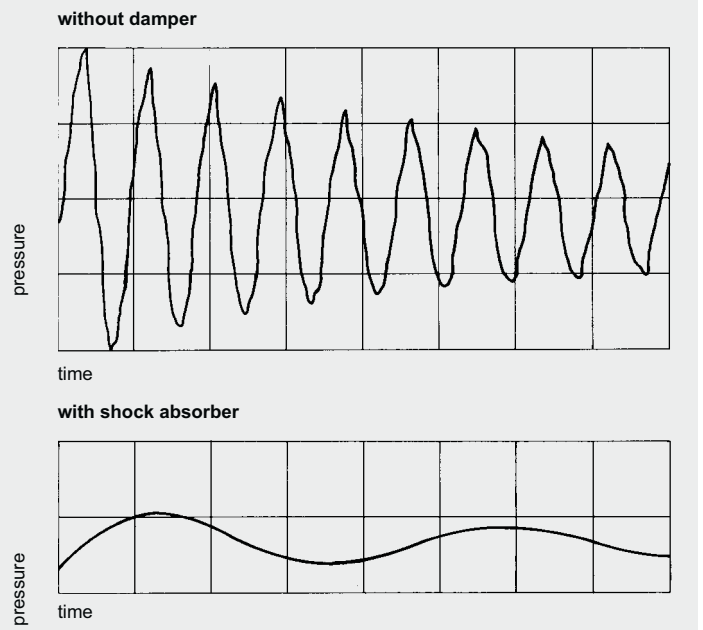
Mode of operation
Trouble-free pump operation is only possible if no cavitation occurs in the pump suction and pipe oscillations are prevented.

A relatively high fluid volume in the suction flow stabiliser in relation to the displacement volume of the pump reduces the acceleration effects of the fluid column in the suction line. Also an air separation is achieved due to the extremely low flow rate in the suction flow stabiliser and the deflection on a baffle. By adjusting the charging pressure of the bladder to the operating conditions, the best possible pulsation damping is achieved.

Construction
The HYDAC suction flow stabiliser consists of a welded vessel in steel or stainless steel.
Inlet and outlet are on opposite sides and are separated by a baffle. The upper part houses the encapsulated bladder. In addition, there is a vent screw in the cover plate and a drainage facility on the bottom.

Installation
As close as possible to the suction inlet of the pump. Mounting position vertical (gas valve pointing upwards).

1.2.3 Shock absorber Type SB...A



General
The HYDAC shock absorber

- reduces pressure shocks;
- protects pipelines and valves from being destroyed.

Applications
The accumulators are particularly suitable for use in pipelines with quick-acting valves or flaps and whilst pumps are being switched on and off.

They are also suitable for energy storage in low pressure applications.

Mode of operation
Sudden changes in pipeline flow, such as those caused by pump failure or the closing or opening of valves, can cause pressures which are many times higher than the normal values.

The shock absorber prevents this by converting potential into kinetic energy and vice versa. This prevents pressure shocks and protects pipelines, valves, control instruments and other devices from destruction.

Construction
The HYDAC shock absorber consists of:

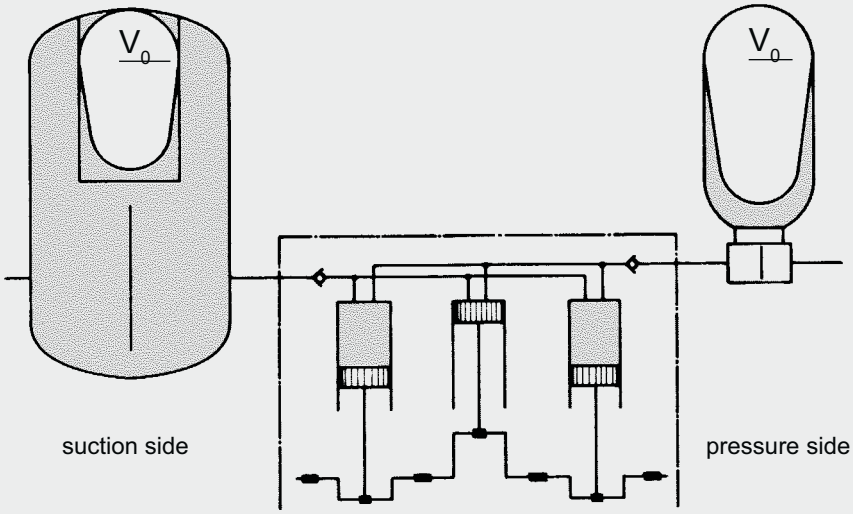
- the welded pressure vessel in carbon steel with or without corrosion protection or in stainless steel;
- the connection including perforated disc which prevents the flexible bladder from extruding from the vessel, and the flange;
- the bladder in various compounds as shown under point 1.4.1 with built-in gas valve, which is used for charging pressure p_0 and for possible monitoring activities.

Special model
Shock absorbers can also be in the form of diaphragm or piston accumulators. Available on request.

Installation
As close as possible to the source of the erratic condition. Mounting position vertical (gas valve pointing upwards).

1.3 SIZING

1.3.1 Pulsation damper and suction flow stabiliser



On the suction and pressure side of piston pumps almost identical conditions regarding non uniformity of the flow rate occur. Therefore the same formulae for determining the effective gas volume are used for calculating the damper size. That in the end two totally different damper types are used is due to the different acceleration and pressure ratios on the two sides.

Not only is the gas volume V_0 a decisive factor but also the connection size of the pump has to be taken into account when selecting the pulsation damper.
In order to avoid additional variations in cross-section which represent reflection points for vibrations, and also to keep pressure drops to a reasonable level, the connection cross-section of the damper must be the same as the pipeline.

The gas volume V_0 of the damper is determined with the aid of the formula for adiabatic changes of state.

By giving the residual pulsation or the gas volume, the damper size can be calculated with the aid of the HYDAC software **ASP** (**A**ccumulator **S**imulation **P**rogram). The results can then be printed out or the data files can be stored in ASP format.

The ASP-program is available free of charge via our website www.hydac.com or via e-mail to speichertechnik@hydac.com.

Designations:

ΔV = fluctuating fluid volume [L]

q = piston stroke volume [L]

$$q = \frac{\pi \cdot d_k^{\,2}}{4} \cdot h_k$$

d_k = piston diameter [dm]

h_k = piston stroke [dm]

δ = coefficient of cyclic variation of the pump

z = no. of compressions / effective cylinders per revolution

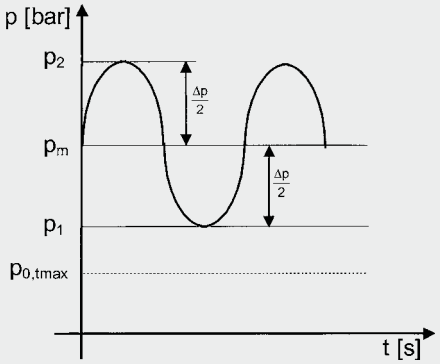
x = residual pulsation [± %]

κ = isentropic exponent

Φ = pressure ratio of pre-charge pressure to operating pressure [0.6 ... 0.9]

$$\Phi = \frac{p_0}{p_m}$$

Δp = height of pressure fluctuations $\Delta p = p_2 - p_1$ [bar]



Formulae:

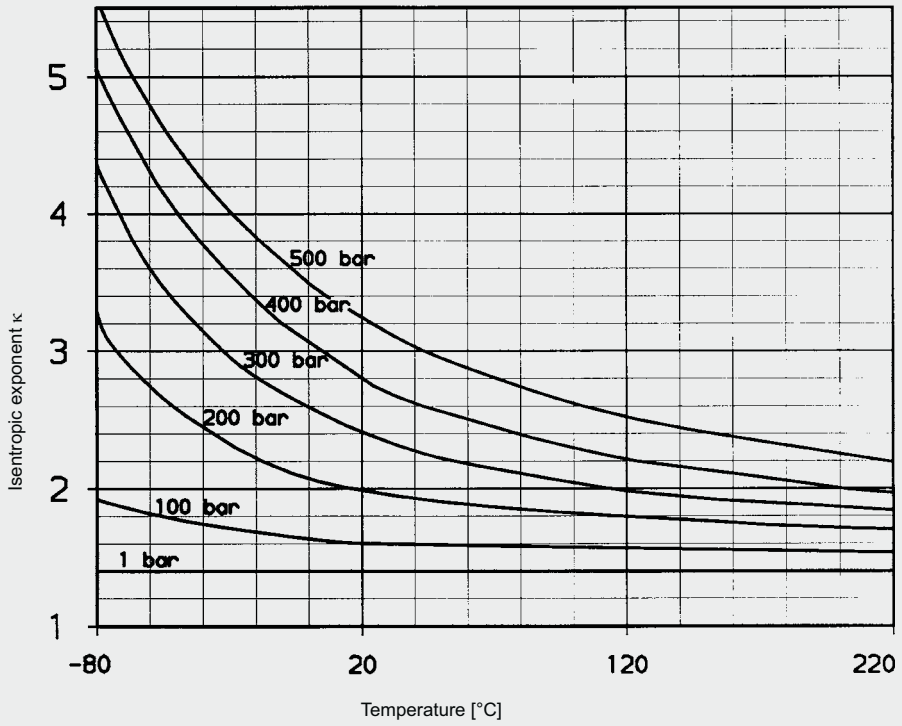
$$V_0 = \frac{\Delta V}{\left[\frac{\Phi}{1-\frac{x}{100}}\right]^{\frac{1}{\kappa}} - \left[\frac{\Phi}{1+\frac{x}{100}}\right]^{\frac{1}{\kappa}}}$$

$$\Delta V = \delta \cdot q$$

$$x[\pm\,\%] = \left|\frac{p_1 - p_m}{p_m} \cdot 100\right|$$

$$= \left|\frac{p_2 - p_m}{p_m} \cdot 100\right|$$

Isentropic exponent κ dependent on pressure and temperature:



δ-values for piston pump
(others on request):

z	δ- value single acting	δ- value double acting
1	0.60	0.25
2	0.25	0.20
3	0.13	0.07
4	0.12	0.05
5	0.05	0.02
6	0.13	0.07
7	0.02	0.01
8	0.08	0.08
9	0.01	0.01

Calculation example

Parameters:

Single-acting 3-piston pump
Piston diameter: 70 mm
Piston stroke: 100 mm
Rpm: 370 min^{-1}
Flow rate: 244 l/min
Operating temperature: 20 °C
Operating pressure
– pressure side: 200 bar
– suction side: 4 bar

Required:

a) Suction flow stabiliser for a residual pulsation of ± 2.5%

b) Pulsation damper for a residual pulsation of ± 0.5%

Solution:

a) Determining the required suction flow stabiliser

$$V_0 = \frac{\Delta V}{\left[\frac{\Phi}{1-\frac{x}{100}}\right]^{\frac{1}{\kappa}} - \left[\frac{\Phi}{1+\frac{x}{100}}\right]^{\frac{1}{\kappa}}}$$

$$V_0 = \frac{0{,}13 \cdot \frac{\pi \cdot 0{,}7^2}{4} \cdot 1{,}0}{\left[\frac{0{,}6}{1-\frac{2{,}5}{100}}\right]^{\frac{1}{1{,}4}} - \left[\frac{0{,}6}{1+\frac{2{,}5}{100}}\right]^{\frac{1}{1{,}4}}}$$

V_0 = 2.0 l

Selected: SB16S-25

b) Determining the required pulsation damper

$$V_0 = \frac{\Delta V}{\left[\frac{\Phi}{1-\frac{x}{100}}\right]^{\frac{1}{\kappa}} - \left[\frac{\Phi}{1+\frac{x}{100}}\right]^{\frac{1}{\kappa}}}$$

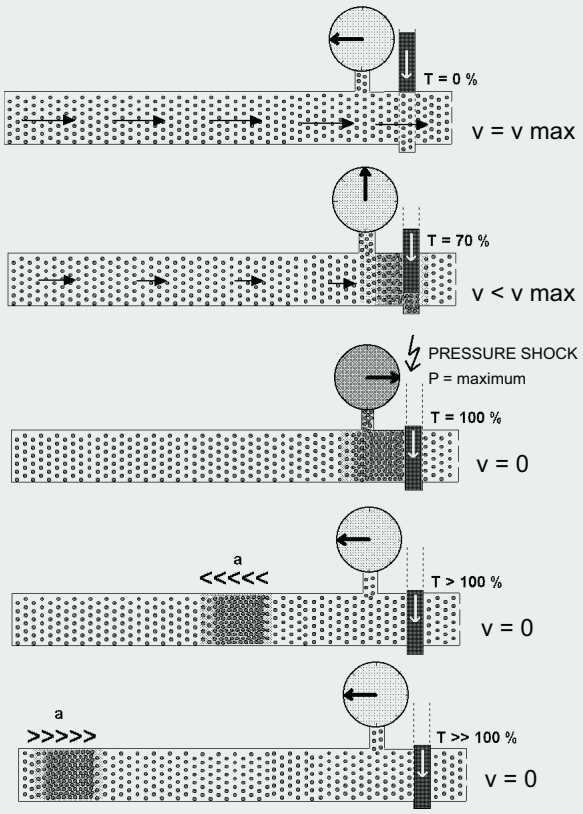
$$V_0 = \frac{0{,}13 \cdot \frac{\pi \cdot 0{,}7^2}{4} \cdot 1{,}0}{\left[\frac{0{,}7}{1-\frac{0{,}5}{100}}\right]^{\frac{1}{2{,}0}} - \left[\frac{0{,}7}{1+\frac{0{,}5}{100}}\right]^{\frac{1}{2{,}0}}}$$

V_0 = 11.9 L

Selected: SB330P-13

1.3.2 **Shock absorber**

Pressure shock produced when a valve is closed without a hydraulic accumulator



Simplified pressure shock calculation for the closing of a valve

Estimate of Joukowsky's max. occurring pressure shock

$\Delta p(N/m^2) = \rho \cdot a \cdot \Delta v$

ρ (kg/m³) = fluid density

$\Delta v = v - v_1$

Δv = change of the fluid speed

v (m/s) = speed of the fluid before the change in its condition

v_1 (m/s) = speed of the fluid after the change in its condition

a (m/s) = velocity of the pressure wave propagation

$$a\ (m/s) = \frac{1}{\sqrt{\rho \cdot \left[\frac{1}{K} + \frac{D}{E \cdot e}\right]}}$$

K (N/m²) = compression modulus of the fluid

E (N/m²) = modulus of elasticity of the pipeline

D (mm) = internal diameter of pipeline

e (mm) = wall thickness of pipeline

The pressure wave runs to the other end of the pipeline and will reach the valve again after time t (reflection time), whereby:

$$t\ (s) = \frac{2 \cdot L}{a}$$

L (m) = length of the pipeline

T (s) = effective operating time (closing) of the valve

If T < t then:

$p_{max} = p_1 + \Delta p$

If T > t then:

$$p_{max} = p_1 + \rho \cdot a \cdot \Delta v \cdot \frac{t}{T}$$

Determining the required damper size

The accumulator must absorb the kinetic energy of the fluid by converting it into potential energy within the pre-determined pressure range. The change of state of the gas is adiabatic in this case

$$V_0 = \frac{m \cdot v^2 \cdot 0{,}4}{2 \cdot p_1 \cdot \left[\left(\frac{p_2}{p_1}\right)^{1-\frac{1}{\kappa}} - 1\right] \cdot 10^2} \cdot \left(\frac{p_1}{p_0}\right)^{\frac{1}{\kappa}}$$

m (kg) = weight of fluid in the pipeline

v (m/s) = speed of fluid

p_1 (bar) = zero feed height of the pump

p_2 (bar) = permitted operating pressure

p_0 (bar) = pre-charge pressure

A special calculation program to analyse the pressure curve is available for sizing during pump failure or start-up and for manifolds.

Calculation example
Rapid closing of a shut-off valve in a refuelling line.

Parameters:

Length of the pipe line L:
2000 m

NW of pipeline D:
250 mm

Wall thickness of pipeline e:
6.3 mm

Material of pipeline:
Steel

Flow rate Q:
432 m³/h = 0.12 m³/s

Density of medium ρ:
980 kg/m³

Zero feed height of pump p_1:
6 bar

Min. operating pressure p_{min}:
4 bar

Effective closing time of the valve T:
1.5 sec.
(approx. 20% of the total closing time)

Operating temperature:
20 °C

Compression modulus of the fluid K:
1.62×10^9 N/m²

Elasticity modulus (steel) E:
2.04×10^{11} N/m²

Required:

Size of the required shock absorber, when the max. pressure (p_2) must not exceed 10 bar.

Solution:

Determination of reflection time:

$$a = \frac{1}{\sqrt{\rho \cdot \left[\frac{1}{K} + \frac{D}{E \cdot e}\right]}}$$

$$a = \frac{1}{\sqrt{980 \cdot \left[\frac{1}{1.62 \cdot 10^9} + \frac{250}{2.04 \cdot 10^{11} \cdot 6.3}\right]}}$$

$$a = 1120 \ \text{m/s}$$

$$t = \frac{2 \cdot L}{a} = \frac{2 \cdot 2000}{1120} = 3.575 \ \text{s} \ *$$

* since T < t the max. pressure surge occurs and the formula as shown in Point 1.3.2 must be used.

$$v = \frac{Q}{A}$$

$$v = \frac{0.12}{0.25^2 \cdot \pi/4} = 2.45 \ \text{m/s}$$

$$\Delta p = \rho \cdot a \cdot \Delta v$$

$$\Delta p = 980 \cdot 1120 \cdot (2.45\text{-}0) \cdot 10^{-5}$$

$$= 26.89 \ \text{bar}$$

$$p_{max} = p_1 + \Delta p$$

$$p_{max} = 6 + 26.89 = 32.89 \ \text{bar}$$

Determining the required gas volume:

$$p_0 \leq 0.9 \cdot p_{min}$$

$$p_0 \leq 0.9 \cdot 5 = 4.5 \ \text{bar}$$

$$V_0 = \frac{m \cdot v^2 \cdot 0.4}{2 \cdot p_1 \cdot \left[\left[\frac{p_2}{p_1}\right]^{1-\frac{1}{\kappa}} - 1\right] \cdot 10^2} \cdot \left[\frac{p_1}{p_0}\right]^{\frac{1}{\kappa}}$$

with $m = V \cdot \rho = \frac{\pi}{4} \cdot D^2 \cdot L \cdot \rho$

$$V_0 = \frac{\frac{\pi}{4} \cdot 0.25^2 \cdot 2000 \cdot 980 \cdot 2.45^2 \cdot 0.4}{2 \cdot 7 \cdot \left[\left[\frac{11}{7}\right]^{1-\frac{1}{1.4}} - 1\right] \cdot 10^2} \cdot \left[\frac{7}{4.5}\right]^{\frac{1}{1.4}}$$

$$V_0 = 1641 \ \text{l}$$

Selected:

4 off shock absorbers
SB35AH-450.

1.4. TECHNICAL DATA

1.4.1 **MODEL CODE** (also order example)
Pulsation damper, suction flow stabiliser, shock absorber

SB330 P – 10 A 1 / 112 U – 330 Al

Series
SB... = with bladder
SBO... = with diaphragm

Type
A = shock absorber
AH = high flow shock absorber
P = pulsation damper
PH = high flow pulsation damper
S = suction flow stabiliser

Nominal volume [l]

Fluid connection
A = threaded connection
E = threaded connection for welded construction (diaphragm accumulators only)
F = flange [4)]

Type code
1 = standard model (not for threaded construction)
2 = back-up model [1)]
6 = standard model for thread-type diaphragm accumulators of the type SBO...P-...A6

Material code [2)]
Standard model = 112 for mineral oils
depending on operating medium

Fluid connection
1 = carbon steel
2 = high tensile steel
3 = stainless steel
4 = chemically nickel-plated (internal coating) [1)]
6 = low temperature steel

Accumulator shell
0 = plastic (internal coating) [1)]
1 = carbon steel
2 = chemically nickel-plated (internal coating) [1)]
4 = stainless steel [1)]
6 = low temperature steel

Accumulator bladder/diaphragm [3)]
2 = NBR20 (acrylonitrile butadiene)
3 = ECO (ethylene oxide epichlorohydrin)
4 = IIR (butyl)
5 = NBR21 (low temperature NBR)
6 = FKM (fluoro rubber)
7 = other (e.g. PTFE, EPDM)

Certification code [2)]
U = PED 97/23/EC

Permitted operating pressure [bar]

Connection thread to
Al = ISO 228 (BSP), standard connection
Bl = DIN 13 to ISO 965/1 (metric) [4)]
Cl = ANSI B1.1 (UNF thread, sealing to SAE standard) [4)]
Dl = ANSI B1.20 (NPT thread) [4)]

SBO250P-0.075E1 and for SBO210P-0.16E1:
AK = ISO 228 (BSP), standard connection

[1)] Not available for all models
[2)] Not all combinations are possible
[3)] When ordering spare bladders, please state smallest bladder connection port size at gas charging end
[4)] Please give full details when ordering

1.4.2 General

Operating pressure
See tables (may differ from nominal pressure for foreign test certificates).

Nominal volumes
See tables

Effective gas volume
See tables, based on nominal dimensions. This differs slightly from the nominal volume and must be used when calculating the usable volume.

On the diaphragm accumulator, the effective gas volume corresponds to the nominal volume.

Usable volume
Volume of fluid which is available between the operating pressures p_2 and p_1.

Fluids
Mineral oils, hydraulic oils, non-flam fluids, water, emulsions, fuels.
Other fluids on request.

Gas charge
When supplied, the accumulator is only pre-charged for storage purposes. Higher pre-charge pressures are possible by arrangement.

Hydraulic accumulators must only be charged with nitrogen.
Never use other gases.
Risk of explosion!

Permitted operating temperature
-10 °C ... +80 °C
263 K ... 353 K
for material code 112.

Others on request.

Permitted pressure ratio
Ratio of maximum operating pressure p_2 to gas pre-charge pressure p_0.

See Catalogue section:

- Accumulators
  No. 3.000

General safety instructions
On no account must any welding, soldering or mechanical work be carried out on the accumulator shell.

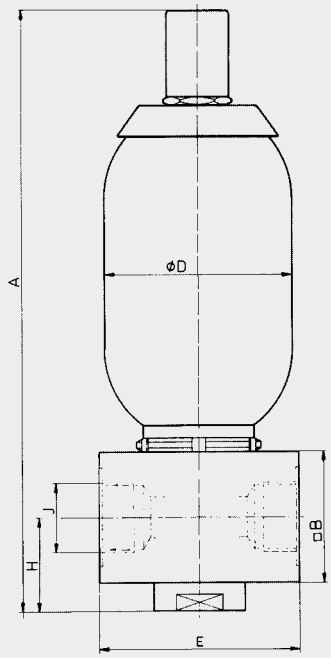
After the hydraulic line has been connected it must be completely vented. Work on systems with hydraulic dampers (repairs, connecting pressure gauges etc) must only be carried out once the pressure and the fluid have been released.

Please read the operating manual!

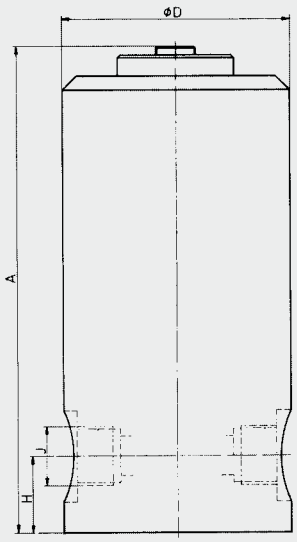
- Bladder Accumulators
  No. 3.201.CE
- Diaphragm Accumulators
  No. 3.100.CE
- Piston Accumulators
  No. 3.301.CE

1.4.3. **Pulsation damper**

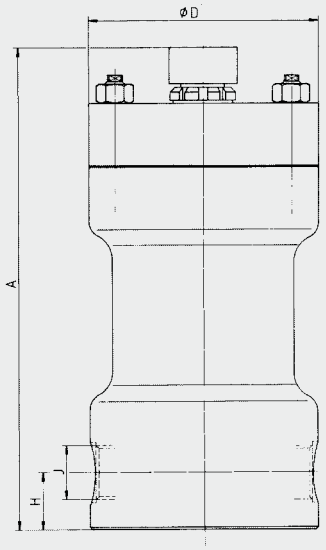
SB330/550P(PH)-...



SB800P-...



SB1000P-...

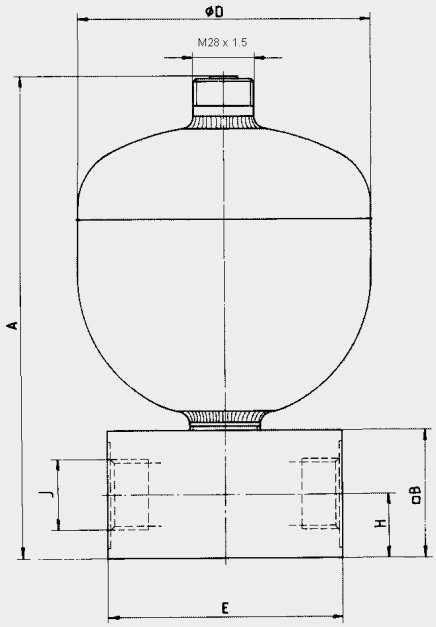


Dimensions SB

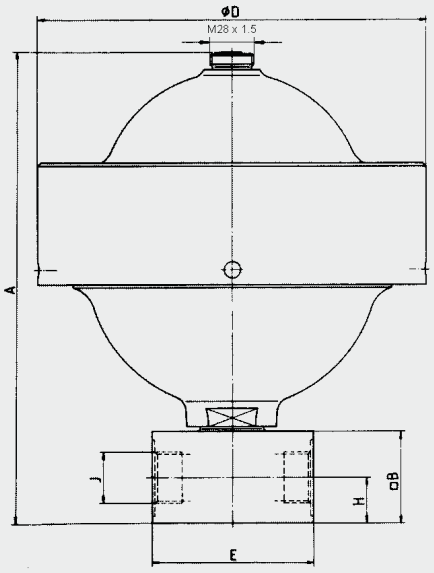
Nominal volume [l]	Max. operating pressure* [bar]	Effective gas volume [l]	Weight [kg]	A [mm]	□ B [mm]	Ø D [mm]	E [mm]	H [mm]	J [2] thread ISO 228	Series
1	330	1.0	11	365	80	118	120	57	G 1 1/4	SB330P
	550		13	384	70	121		53		SB550P
1.5	800	1.3	36	346	–	160	–	55		SB800P
	1000		94	414	–	215	–	49	[1]	SB1000P
2.5	330	2.4	16	570	80	118	120	57	G 1 1/4	SB330P
	550	2.5	20	589	70	121		53		SB550P
4	330	3.7	18	455	80	171		57		SB330P
			26	491	100		150	85	G 1 1/2	SB330PH
5	550	4.9	26	917	70	121	120	53	G 1 1/4	SB550P
6	330	5.7	20	559	80	171		57		SB330P
			28	593	100		150	85	G 1 1/2	SB330PH
10		9.3	40	620		229				SB330P
			50	652	130x140			100	SAE2"-6000 PSI	SB330PH
13	330	12.0	48	712	100			85	G 1 1/2	SB330P
20		18.4	70	920						SB330P
			80	952	130x140			100	SAE2"-6000 PSI	SB330PH
24	330	23.6	82	986	100			85	G 1 1/2	SB330P
32		33.9	100	1445						SB330P
			110	1475	130x140			100	SAE2"-6000 PSI	SB330PH

* Certification to PED 97/23/EC
[1] M56x4, high pressure connection DN 16, others on request
[2] Standard connection code = AI, others on request

SBO...P...E



SBO...P...A6



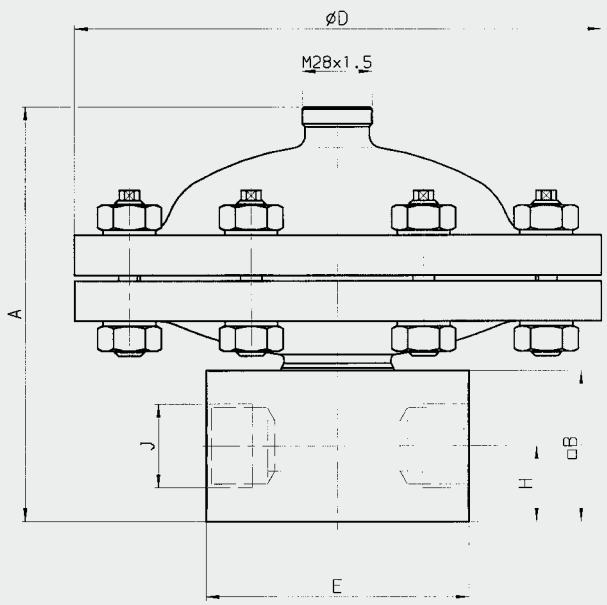
Dimensions SBO

Nominal volume	Max. operating pressure*		Weight	A	□ B	Ø D	E	H	J [1] thread	Series	
[l]	carbon steel [bar]	stainl. st. [bar]	[kg]	[mm]	[mm]	[mm]	[mm]	[mm]	ISO 228		
0.075	250	–	0.9	131	–	64	hex. 41	13	G 1/4	SBO250P-...E1	weld-type
0.16	210	180	1.0	143	–	74				SBO210P-...E1	
0.32		160	2.6	175	50	93	80	25	G 1/2		
0.5		–	3.0	192		105					
0.6	330	–	5.6	222	60	115	105	30	G 1	SBO330P-...E1	
0.75	210	140	5.1	217		121				SBO210P-...E1	
1.0	200	–	6.0	231		136				SBO200P-...E1	
1.4	140	–	6.2	244		145				SBO140P-...E1	
	210	–	7.7	250		150				SBO210P-...E1	
	250	–	8.2	255		153				SBO250P-...E1	
2.0	100	100	6.3	261		160				SBO100P-...E1	
	210	–	8.9	267		167				SBO210P-...E1	
3.5	250	–	13.5	377		170				SBO250P-...E1	
4.0	–	50	7.9	368		158				SBO50P-...E1	
		250	13.5	377		170				SBO250P-...E1	
0.25	500	350	4.5 (5.2)	162 (180)	50	115	80	25	G 1/2	SBO500P-...A6	thread-type
0.6	330	250	8.9 (8.4)	202 (215)	60	140 (142)	95		G 1	SBO450P-...A6	
1.3	400	–	13.8	267		199	105	30		SBO400P-...A6	
2.0	250	180	15.6 (15)	285 (274)		201 (199)				SBO250P-...A6	
2.8	400	–	24.6	308		252				SBO400P-...A6	
4.0		–	36.6	325		287					

* certification to PED 97/23/EC
[1] Standard connection code = AI, others on request
( ) brackets indicate different dimensions for stainless steel version

Pulsation dampers for aggressive media

SBO...P-...A6/347...(PTFE)



Pulsation damper in stainless steel with PTFE coated diaphragm and PTFE or FFKM seals.
Also available without connection block.

Certification:
PED 97/23/EC

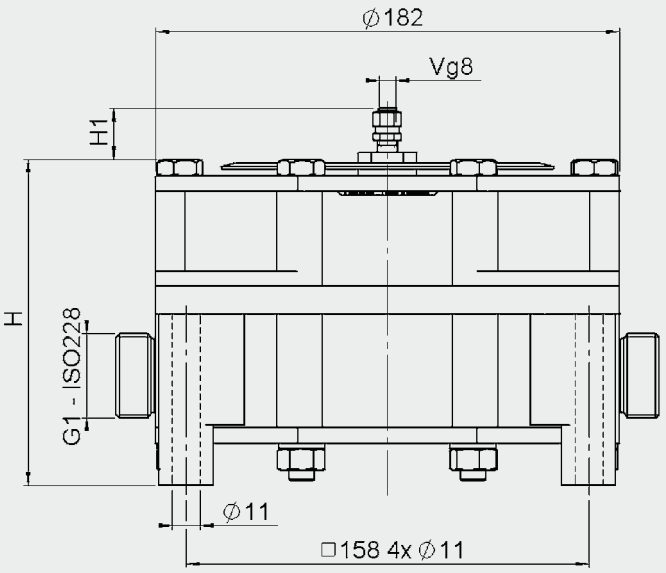
Permitted operating temperature:
-15 °C ... +80 °C

Permitted pressure ratio $p_2 : p_0 = 2 : 1$

Nominal volume [l]	Max. operating pressure [bar]	Weight [kg]	A [mm]	□ B [mm]	Ø D [mm]	E [mm]	H [mm]	J [1] thread ISO 228
0.2	40	11	140	60	210	105	30	G 1
	250	27	197		230			
0.5	40	12	165		210			
	250	26	200		230			

[1] Standard connection code = AI, others on request

SBO...P-...A4/777... (PVDF/PTFE)



Pulsation damper in PVDF with PTFE-coated diaphragm.

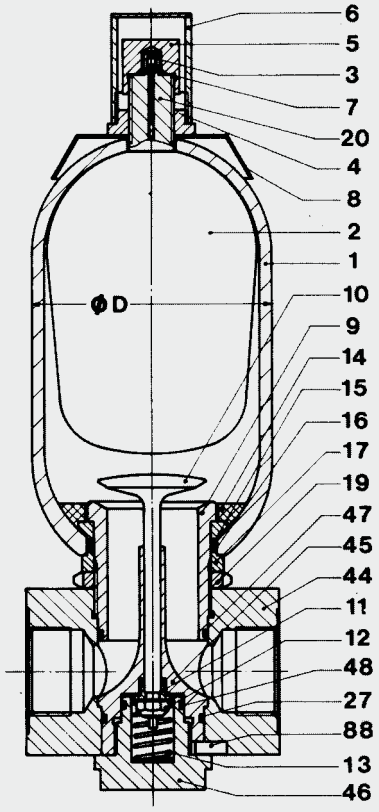
Permitted operating temperature:
-15 °C ... +65 °C

Permitted pressure ratio $p_2 : p_0 = 2 : 1$

Nominal volume [l]	Max. operating pressure [bar]	Weight [kg]	H [mm]	H1 [mm]
0.2	10	5.7	128	20
	16	6.5	130	18
0.5	10	6.0	168	20
	16	6.8	170	19

Spare parts

SB...P



Description	Item
Gas valve insert*	3
Bladder, complete* consisting of:	
Bladder	2
Gas valve insert	3
Retaining nut	4
Cap nut	5
Valve protection cap	6
O-ring	7
Anti-extrusion ring*	14
Seal kit* consisting of:	
O-ring	7
Washer	15
O-ring	16
Support ring	23
O-ring	27
O-ring	47
O-ring	48

* recommended spares

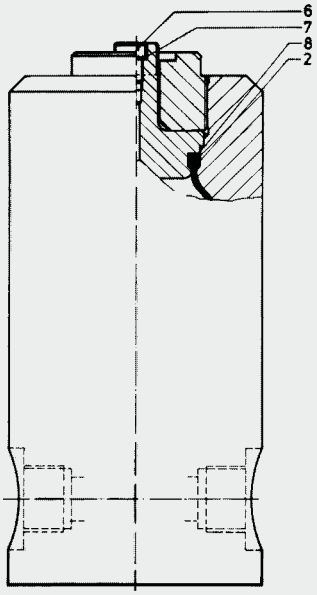
Description	Item
Connection, complete consisting of:	
Oil valve body	9
Valve plate	10
Damping sleeve	11
Lock nut	12
Spring	13
Anti-extrusion ring	14
Washer	15
O-ring	16
Spacer	17
Lock nut	19
Support ring (only for 330 bar)	23
O-ring	27
Connector	44
Guide piece	45
Cap	46
O-ring	47
O-ring	48
Locking key	88

O-ring dimensions (mm)

Series	Nom. volume	Item 7	Item 16	Item 27	Item 47	Item 48
SB330P	1- 6 l	7.5x2	55x3.5 [1)]	42.2x3 [1)]	46x3 [1)]	24.2x3 [1)]
SB550P	1- 5 l	7.5x2	50.17x5.33 [1)]	37.82x1.78 [1)]	40.94x2.62 [1)]	23.52x1.78 [1)]
SB330P/PH	10-32 l/4+6 l	7.5x2	80x5 [1)]	57.2x3 [1)]	67.2x3 [1)]	37.2x3 [1)]
SB330PH	10-32 l	7.5x2	100x5 [1)]	64.5x3 [1)]	84.5x3 [1)]	44.2x3 [1)]

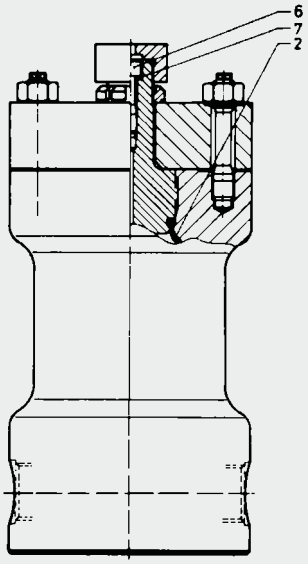
[1)] For code 663 and 665 different dimensions

SB800P



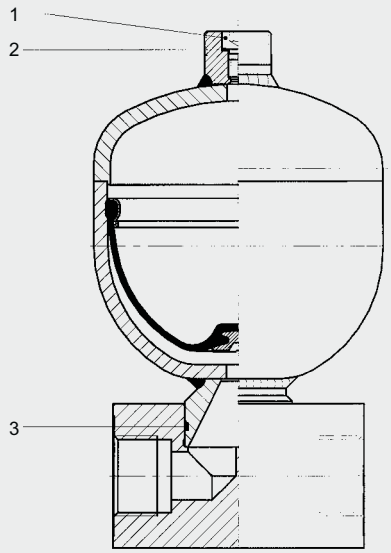
Description	Item
Bladder	2
Charging screw	6
Seal ring U 9.3x13.3x1	7
Support ring	8

SB1000P



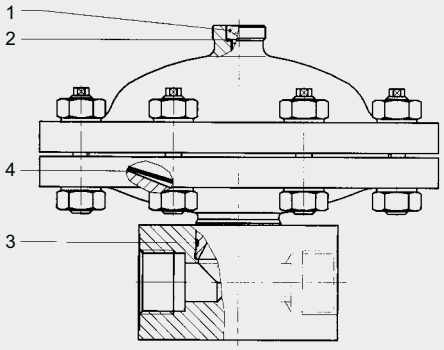
Description	Item
Bladder	2
Charging screw	6
Seal ring	7

SBO...P...E



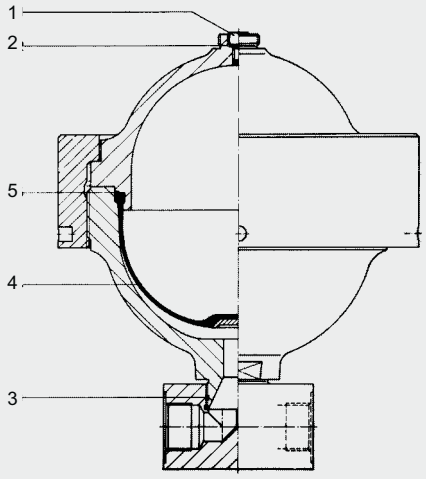
Description	Item
Charging screw	1
Seal ring	2
Seal ring	3

SBO...P-...A6/347...(PTFE)



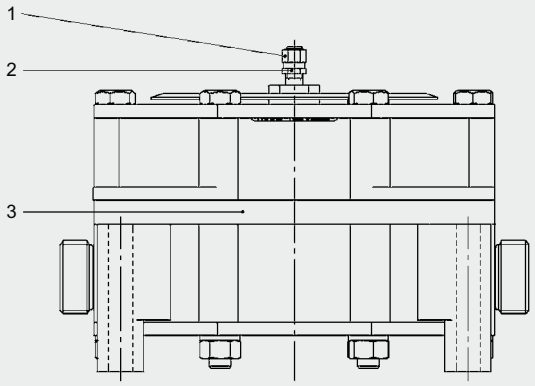
Description	Item
Charging screw	1
Seal ring	2
Seal ring	3
Diaphragm	4

SBO...P...A6



Description	Item
Charging screw	1
Seal ring	2
Seal ring	3
Diaphragm	4
Support ring	5

SBO...P-...A4/777... (PVDF/PTFE)

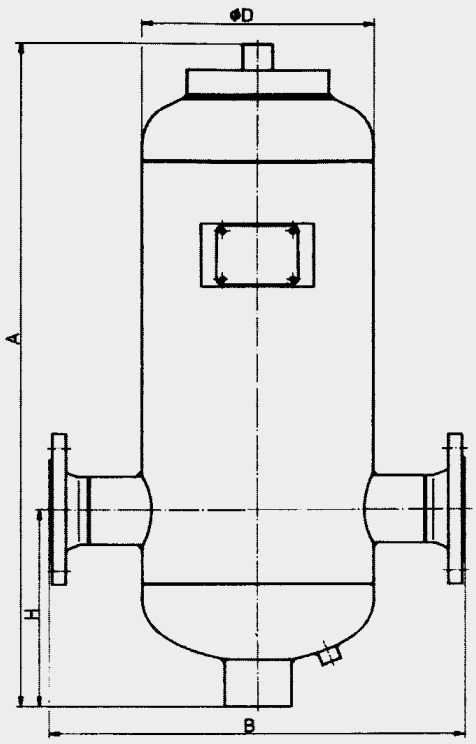


Description	Item
Gas valve complete	1
Gas valve insert brass / stainless steel	2
Diaphragm	3

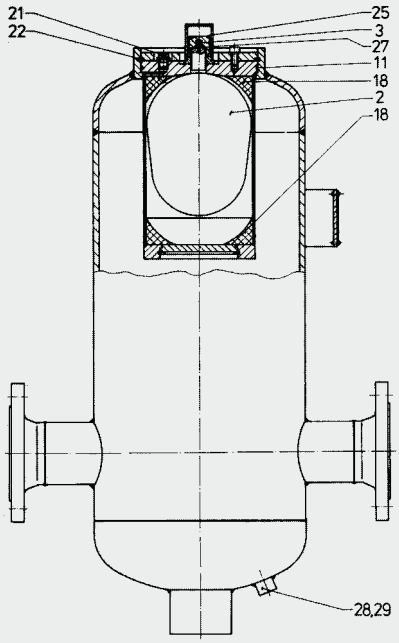
Please read operating manual!
Available on request!

1.4.4 Suction flow stabiliser

SB16S



Spare parts



Dimensions

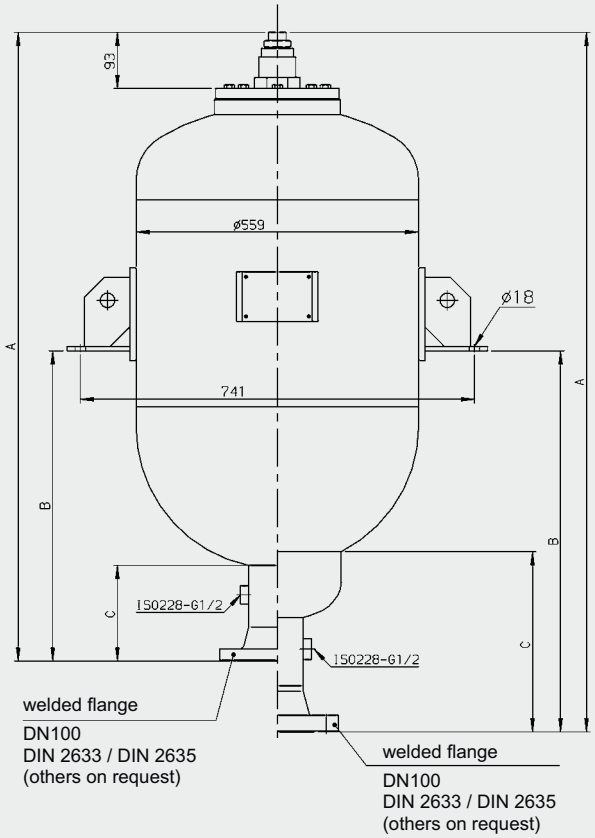
SB 16 S - permitted working pressure 16 [bar]; certified to PED 97/23/EC								
Nominal volume [l]	Fluid volume [l]	Effective gas volume [l]	Weight [kg]	A [mm]	B [mm]	Ø D [mm]	H [mm]	DN DIN 2633
12	12	1	40	580	425	219	220	65
25	25	2.5	60	1025				
40	40	4	85	890	540	300	250	80
100	100	10	140	1150	650	406	350	100
400	400	35	380	2050	870	559	400	125

Further pressure ranges 25 bar, 40 bar; others on request.
Other fluid volumes on request

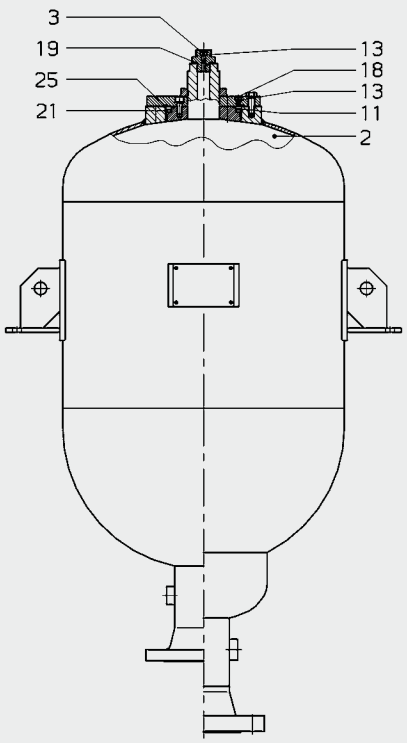
Description	Item
Bladder	2
Gas valve insert	3
O-ring	11
Insertion ring, 2x	18
Lock nut	21
Retaining ring	22
Cap nut	25
O-ring	27
Seal ring	28
Lock nut	29

1.4.5 Shock absorber

SB16/35A(H)



Spare parts



Description	Item
Bladder	2
Lock nut	3
O-ring	11
Seal ring	13
Vent screw	18
O-ring	19
Retaining ring	21
O-ring	25

Dimensions

SB16/35 A - permitted operating pressure 16/35 [bar] (PED 97/23/EC)

Nominal volume	Effective gas volume	Weight [kg]		A (approx.) [mm]		B (approx.) [mm]		C (approx.) [mm]		DN [1] DIN 2633
[l]	[l]	SB16A	SB35A	SB16A	SB35A	SB16A	SB35A	SB16A	SB35A	
100	99	84	144	870	880	390	403	185	198	100
150	143	101	161	1070	1080	490	503			
200	187	122	223	1310	1320	685	698			
300	278	155	288	1710	1720	975	988			
375	392	191	326	2230	2240	1250	1263			
450	480	237	386	2625	2635	1465	1478			

SB16/35 AH - Permitted operating pressure 16/35 [bar] (PED 97/23/EC)

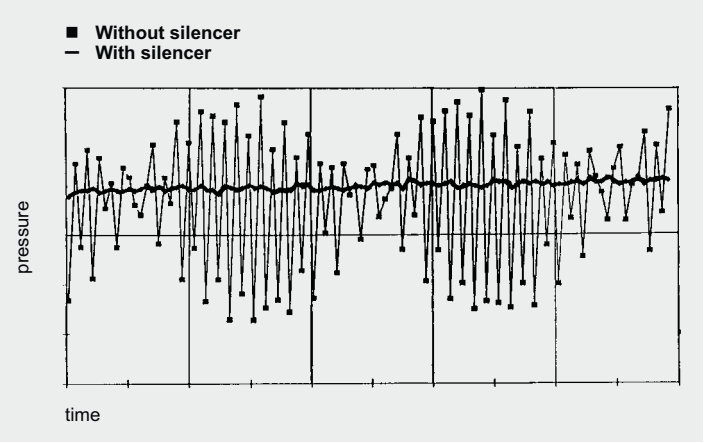
Nominal volume	Effective gas volume	Weight [kg]		A (approx.) [mm]		B (approx.) [mm]		C (approx.) [mm]		DN [1] DIN 2635 DIN 2633
[l]	[l]	SB16AH	SB35AH	SB16AH	SB35AH	SB16AH	SB35AH	SB16AH	SB35AH	
100	99	93	153	957	965	457	465	245	254	80
150	143	110	170	1157	1165	557	565			
200	187	131	230	1417	1425	842	850			
300	278	164	297	1865	1873	1092	1100			
375	392	200	335	2307	2315	1342	1350			
450	480	246	395	2702	2710	1542	1550			

[1] other nominal widths on request

2. SILENCER

2.1. APPLICATION

2.1.1 Silencer for fluid noise damping Type SD...



General

All displacement pumps such as axial and radial piston pumps, vane, gear or screw pumps produce volume and pressure fluctuations which are exhibited as vibrations and noises. Noises are not only produced and transmitted by the pump but they are also the result of mechanical vibrations and vibrations caused by the fluid pulsations, which are amplified when transmitted to larger areas. Insulation, the use of flexible hoses and silencer covers can only provide partial solutions to the problem as they do not prevent transmission to other areas.

Applications

Vehicles, machine tools, plastics machinery, aeroplanes, ships, hydraulic power stations and other systems with a large "surface" are all applications where the noise level can be reduced.

Mode of operation

The HYDAC fluid silencer is based on the principle of an expansion chamber with interference line.

By reflecting the oscillations within the SILENCER the majority of the oscillations are damped across a wide frequency spectrum.

Construction

The HYDAC SILENCER consists of a welded or forged external housing, an internal tube and two pipe connections on opposite sides.

The SILENCER has no moving parts and no gas charge and is therefore absolutely maintenance free.

The HYDAC SILENCER can be used for mineral oils, phosphate ester and water glycol. A stainless steel model is available for other fluids.

Special model

SILENCERS can also be in the form of diaphragm or piston accumulators. Available on request.

Installation

It is recommended that one connection side is joined via a flexible hose in order to reduce the transmission of mechanical vibrations.

The mounting position of the damper is optional, whereby the flow direction must be taken into account.

Please read the operating manual!
No. 3.701.CE

2.2. SIZING

2.2.1 Silencer

The sizing calculation of the HYDAC silencer is designed to result in a small unit with the best possible damping. The starting point for the selection table is to determine the level of transmission damping D from 20 dB upwards.

$$D = 20 \cdot \log \frac{\Delta p_0}{\Delta p_m}$$

Δp_o = height of pressure fluctuations without silencer
Δp_m = height of pressure fluctuations with silencer

For the selection of the damper the following has to be taken into account:

1) the size of the silencer body
2) the fundamental frequency f of the pump.
   f = i • n / 60 in Hz
   i = number of displacement elements
   n = rpm in min^{-1}

2.2.2 Calculation example

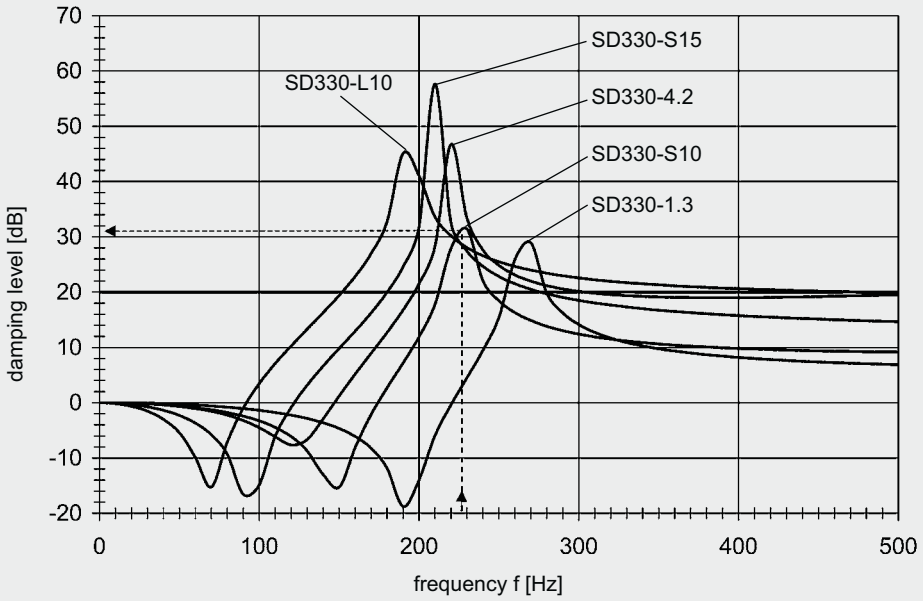
Parameters:

Axial piston pump with 9 pistons
Rpm 1500 min^{-1}
Connection G1
Corresponds to D_j = 19 mm
Flow rate 300 l/min
Operating fluid: mineral oil
Max. operating pressure 210 bar

Solution:

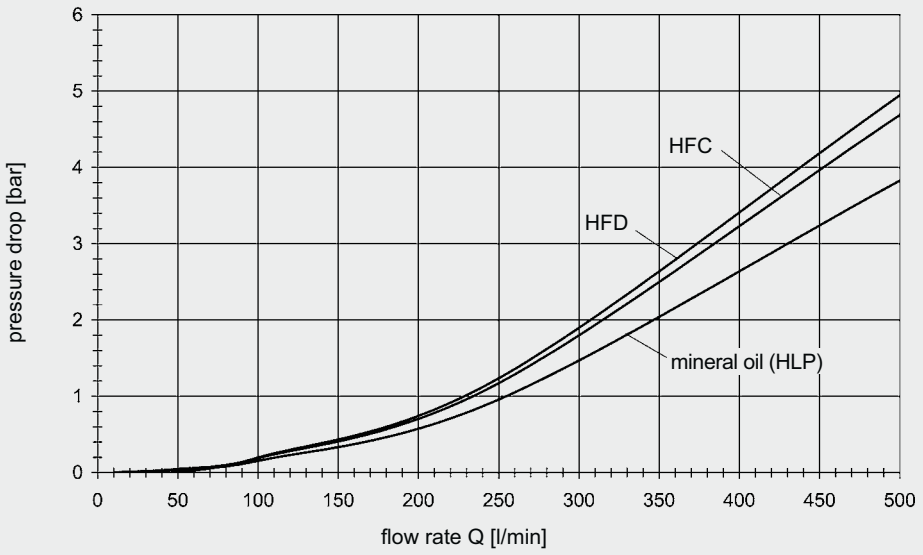
1) Fundamental frequency f
   f = i • n / 60 in Hz
   = 9 • 1500/60
   = 225 Hz
2) From the "Damping curve" graph, the following SILENCER type can be selected:
   SD330-S10/012U-330AE/AE
   transmission damping ≈ 31 dB
   pressure drop ≈ 2 bar.

2.2.3 **Damping curve**

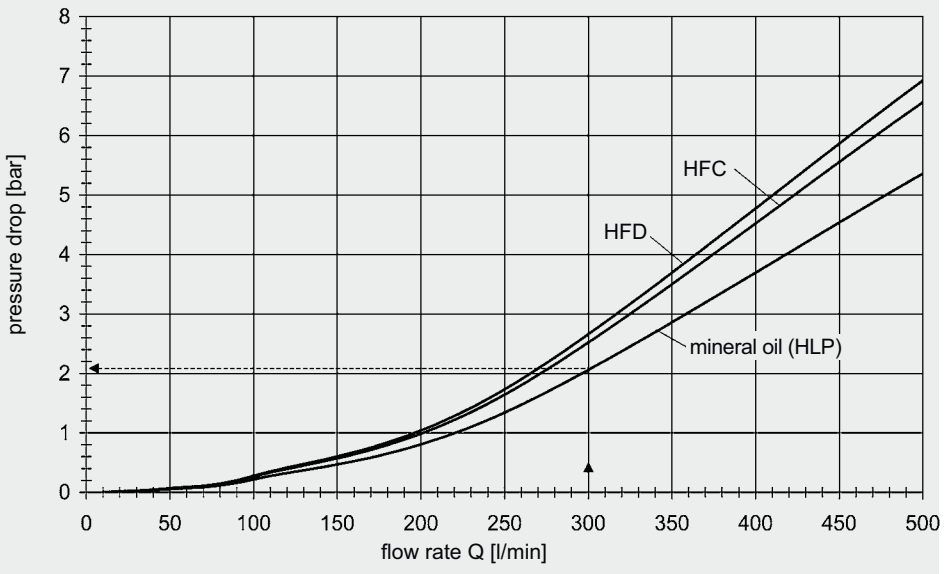


2.2.4 **Pressure drop**

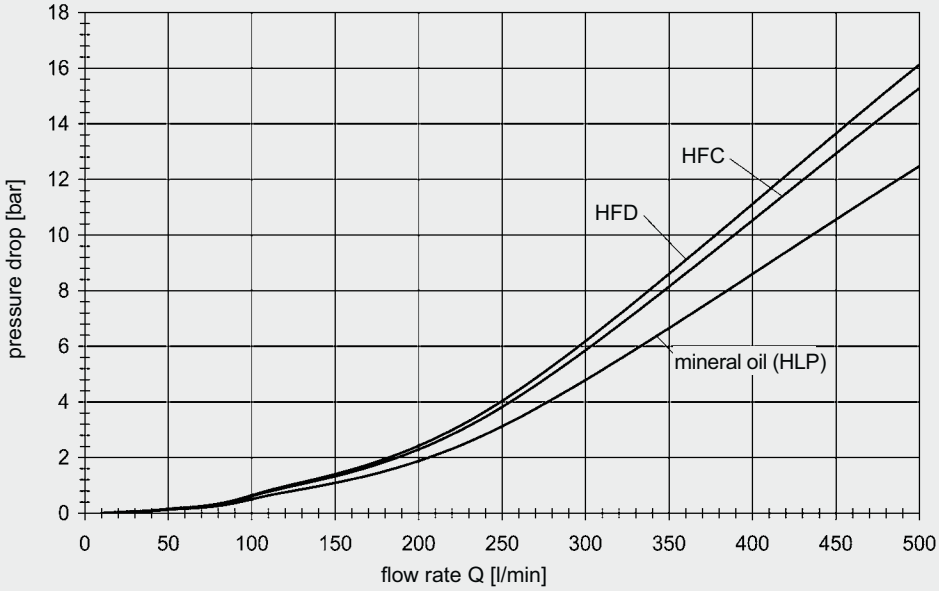
SD330-1.3...



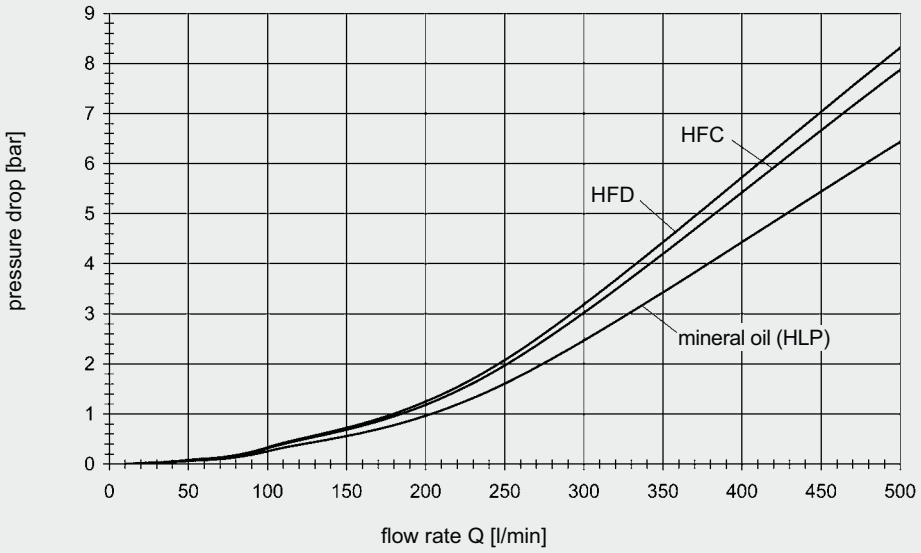
SD330-S10...
SD330-4.2...



SD330-L10...



SD330-S15...



2.3. TECHNICAL DATA

2.3.1 **Model code**
(also order example)

SD330 – S15 / 012 U – 330 AE/AE

Series

Size

Material code

Valve body (not applicable)

Shell
1 = carbon steel
2 = carbon steel with coating*

Seal material
2 = NBR (acrylonitrile butadiene)
6 = FPM (fluoro rubber)

Certification code
U = PED 97/23/EC

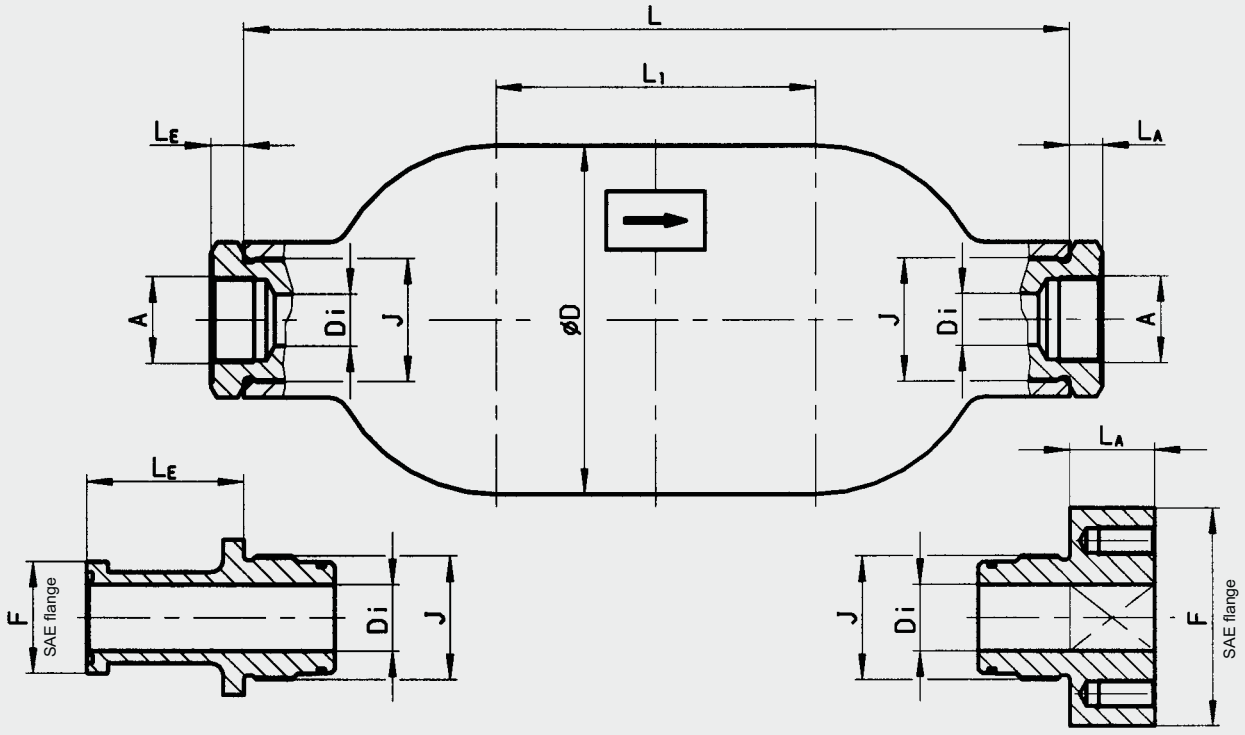
Permitted operating pressure in bar

Inlet connector / Outlet connector
see table 2.3.3

* only on request

2.3.2 **Dimensions**

SD330



Dimensions

Size	Nom. volume [l]	L [mm]	L_1 [mm]	Ø D [mm]	J ISO 228	Weight [kg]
1.3	1.3	250	–	114	G 1	6.5
S 10	1.8	355	155		G 1 1/4	5.5
L 10*	5.5	815	615			14.0
4.2	4.2	346	–	168	G 1 1/2	12.5
S 15	4.7	420	155		G 2	11.4

2.3.3 **Silencer connections**

a) Threaded connection to ISO 228

	Fluid connection A									
	AC G 1/2 - ISO 228 D_i = 13 mm		**AD** G 3/4 - ISO 228 D_i = 16 mm		**AE** G 1 - ISO 228 D_i = 19 mm		**AF** G 1 1/4 - ISO 228 D_i = 25 mm		**AG** G 1 1/2 - ISO 228 D_i = 32 mm	
Size	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]
S 10	13	13	13	13	30	30	33	33	–	
L 10										
S 15	–		16	16	16	16	26	26	36	36

b) Flange connection SAE J518 (Code 62 - 6000 psi)

	Fluid connection F											
	FG SAE 1/2" D_i = 13 mm		**FH** SAE 3/4" D_i = 19 mm		**FI** SAE 1" D_i = 25 mm		**FK** SAE 1 1/4" D_i = 32 mm		**FL** SAE 1 1/2" D_i = 38 mm		**FM** SAE 2" D_i = 50 mm	
Size	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]	L_E [mm]	L_A [mm]
S 10	53	31	59	36	65	36	–		–		–	
L 10												
S 15	–		105		120		76	28	76	28	*	

* on request
– not available

3. NOTE

The information in this brochure relates to the operating conditions and applications described. For applications and operating conditions not described, please contact the relevant technical department. All technical details are subject to change without notice.

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